

TRANSFLUD® industrial & marine





K - CK - CCK FLUID COUPLINGS

DESCRIPTION	pag.	2
PERFORMANCE CURVES		3
STARTING TORQUE CHARACTERISTICS		4
ADVANTAGES		5
STANDARD OR REVERSE MOUNTING		6
PRODUCTION PROGRAM		7 ÷ 8
SPECIAL VERSION (ATEX)		8
SELECTION		9 ÷ 12
DIMENSIONS (IN LINE VERSION)	1	3 ÷ 23
CENTER OF GRAVITY AND MOMENT OF INERTIA		24
DIMENSIONS (PULLEY VERSIONS)	2	25 ÷ 26
SAFETY DEVICES	2	27 ÷ 29
OTHER TRANSFLUID PRODUCTS		30
SALES NETWORK		



1. DESCRIPTION

The TRANSFLUID coupling (K series) is a constant fill type, comprising of three main elements:

- 1 driving impeller (pump) mounted on the input shaft.
- 2 driven impeller (turbine) mounted on the output shaft.
- 3 cover, flanged to the outer impeller, with an oil-tight seal.

The first two elements can work both as pump or turbine.

2. OPERATING CONDITIONS

The TRANSFLUID coupling is a hydrodynamic transmission. The impellers perform asva centrifugal pump and a hydraulic turbine. With an input drive to the pump (e.g. electric motor or Diesel engine) kinetic energy is transferred to the oil in the coupling. The oil is forced, by centrifugal force, across the blades of the pump towards the outside of the coupling.

The turbine absorbs kinetic energy and generates a torque that is always equal to the input torque, thus causing rotation of the output shaft. Since there are no mechanical connections, the wear is practically zero.

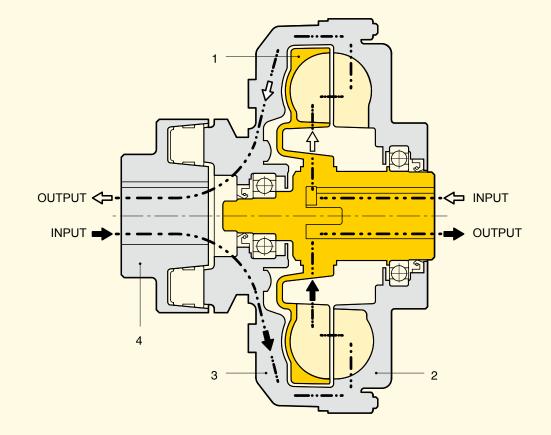
The efficiency is influenced only by the speed difference (slip) between pump and turbine.

The slip is essential for the correct operation of the coupling - there could not be torque transmission without slip! The formula for slip, from which the power loss can be deduced is as follows:

Slip % = $\frac{\text{input speed - output speed}}{\text{input speed}} \times 100$

In normal conditions (standard duty), slip can vary from 1,5% (large power applications) to 6% (small power applications). TRANSFLUID couplings follow the laws of all centrifugal machines:

- transmitted torque is proportional to the square of input speed;
 transmitted power is proportional to the third power of input speed;
- 3 transmitted power is proportional to the fifth power of circuit outside diameter.



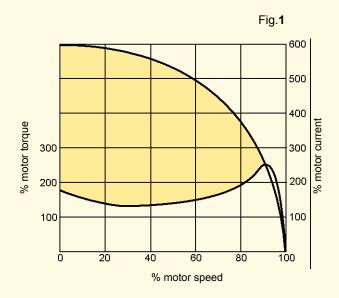
1 - INNER IMPELLER

- 2 OUTER IMPELLER
- 3 COVER

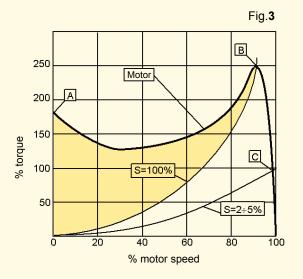
4 - FLEX COUPLING

2.1 Transfluid coupling fitted on electric motors

Three phase asynchronous squirrel cage motors are able to supply maximum torque only, near synchronous speed. Direct starting is the system utilized the most. Figure 1 illustrates the relationship between torque and current. It can be seen that the absorbed current is proportional to the torque only between 85% and 100% of the asynchronous speed.



Any drive system using a Transfluid fluid coupling has the advantage of the motor starting essentially without load. Figure 2 compares the current demands of an electric motor when the load is directly attached versus the demand when a fluid coupling is mounted between the motor and load. The coloured area shows the energy that is lost, as heat, during start-up when a fluid coupling is not used. A Transfluid fluid coupling reduces the motor's current peak during start-up and also reduces the current losses, increasing the lifetime of electric motors. Also at start-up, a fluid coupling allows more torque to pass to the load for acceleration than in drive systems without a fluid coupling.



With a motor connected directly to the load there are the following disadvantages:

- The difference between available torque and the torque required by the load is very low until the rotor has accelerated to between 80-85% of the synchronous speed.
- The absorbed current is high (up to 6 times the nominal current) throughout the starting phase causing overheating of the windings, overloads in the electrical lines and, in cases of frequent starts, major production costs.
- Over-dimensioned motors caused by the limitations indicated above.

To limit the absorbed current of the motor during the acceleration of the load, a ($\lambda \Delta$) (wye - delta) starting system is frequently used which reduces the absorbed current by about 1/3 during starting. Unfortunately, during operation of the motor under the delta configuration, the available torgue is also reduced by 1/3;

and for machines with high inertias to accelerate, overdimensioning of the motor is still required. Finally, this system does not eliminate current peaks originating from the insertion or the commutation of the device.

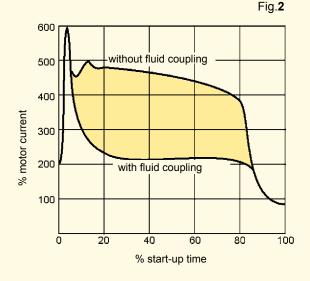


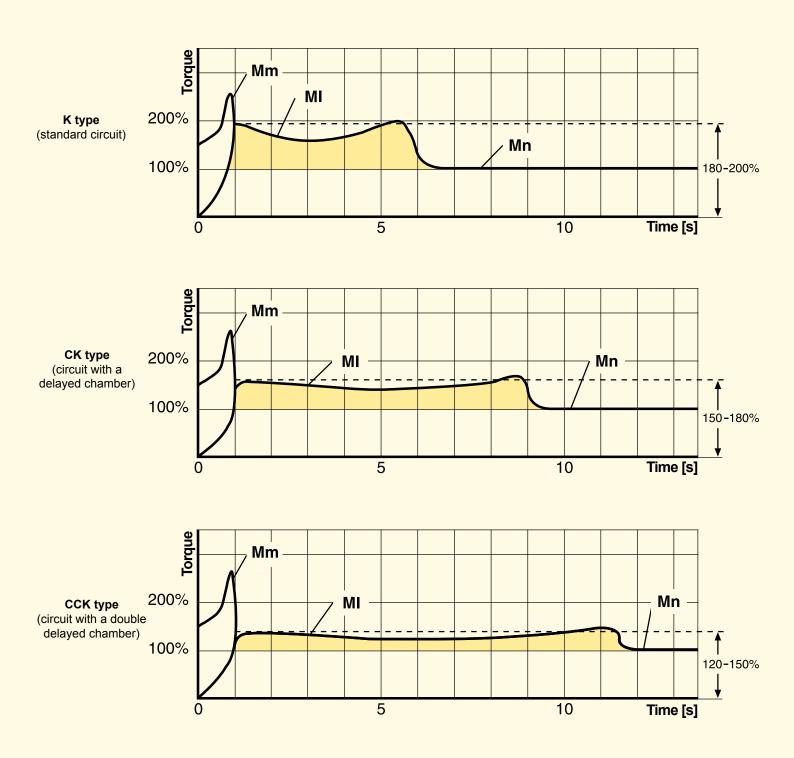
Figure 3 shows two curves for a single fluid coupling and a characteristic curve of an electric motor. It is obvious from the stall curve of the fluid coupling (s = 100%) and the available motor torque, how much torque is available to accelerate the rotor of the motor (colored area). In about 1 second, the rotor of the motor accelerates passing from point A to point B. The acceleration of the load, however, is made gradually by the fluid coupling, utilizing the motor in optimal conditions, along the part of the curve between point B, 100% and point C, 2-5%. Point C is the typical point of operation during normal running.

STARTING TORQUE CHARACTERISTICS



2.2 CHARACTERISTIC CURVES

- MI : transmitted torque from fluid coupling
- Mm : starting torque of the electric motor
- Mn : nominal torque at full load
- : accelerating torque



NOTE: Above starting times are indicative only

3. TRANSFLUID FLUID COUPLINGS WITH A DELAYED FILL CHAMBER

A low starting torque is achieved and with the standard circuit in

a maximum oil fill condition because fluid couplings limit the torque **to less than 200%** of the nominal motor torque. It is possible to limit further the starting torque **down to 160%** of the nominal torque, by decreasing oil fill: however, this creates slip and working temperature increase in the fluid coupling.

The most convenient technical solution is to use fluid couplings with a **delayed fill chamber**, connected to the main circuit by **calibrated bleed orifices. These externally adjustable** valves, available from size **15CK** (Fig. **4b**), can be simply adjusted to vary starting time.

In a standstill position, the **delayed fill chamber** contains part of the filling oil, thus reducing the effective quantity in the working circuit (Fig. **4a**) and a **torque reduction** is obtained, allowing the motor to quickly reach the steady running speed **as if started without load**.

During start-up, oil flows from the **delayed fill chamber** to the main circuit (Fig. **4b**) in a quantity proportional to the rotating speed.

As soon as the fluid coupling reaches the nominal speed, all oil flows into the main circuit (Fig. **4c**) and torque is transmitted with a **minimum slip**.

With a **simple delayed fill chamber**, the ratio between starting and nominal torque may reach **150** %. This ratio may be further reduced down to **120** % with a **double delayed fill chamber**, which contains a higher oil quantity, to be progressively transferred into the main circuit during the starting phase.

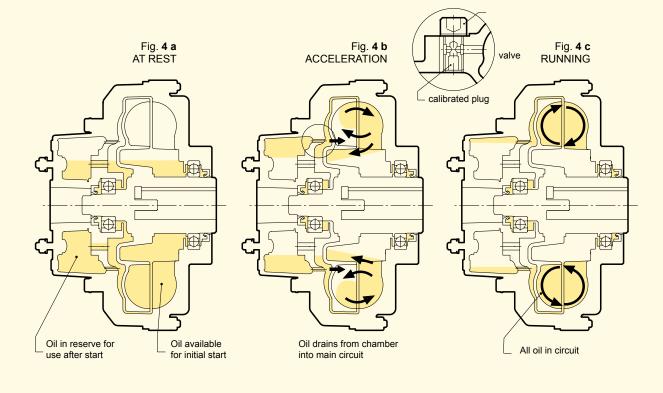
This is ideal for very smooth start-ups with low torque absorptions, as typically required for machinery with large inertia values and for belt conveyors.

The advantages of the **delayed fill chamber** become more and more evident when the power to be transmitted increases.

The simple chamber is available from size 11CK, while the double chamber from size 15CCK.

3.1 SUMMARY OF THE ADVANTAGES GIVEN BY FLUID COUPLINGS

- very smooth start-ups
- reduction of current absorptions during the starting phase: the motor starts with very low load
- protection of the motor and the driven machine from jams and overloads use
- utilization of asynchronous squirrel cage motors instead of special motors with soft starter devices
- higher duration and operating convenience of the whole drive train, thanks to the protection function achieved by the fluid coupling
- higher energy saving, thanks to current peak reduction
- limited starting torque down to 120% in the versions with a double delayed fill chamber
- same torque at input and output: the motor can supply the maximum torque even when load is jammed
- torsional vibration absorption for internal combustion engines, thanks to the presence of a fluid as a power transmission element
- possibility to achieve a high number of start-ups, also with an inversion of the rotation direction
- load balancing in case of a double motor drive: fluid couplings automatically adjust load speed to the motors speed
- high efficiency
- minimum maintenance
- Viton rotating seals
- cast iron and steel material with anticorrosion treatment



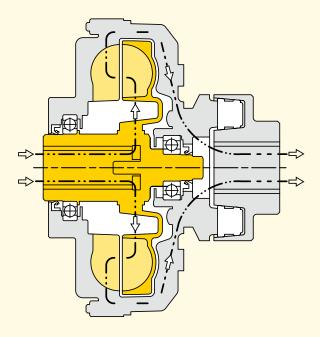
STANDARD OR REVERSE MOUNTING



4. INSTALLATION

4.1 STANDARD MOUNTING

Driver inner impeller



Minimum possible inertia is added to the motor, and therefore free to accelerate more quickly.

During the starting phase, the outer impeller gradually reaches the steady running condition. For very long starting times, heat dissipation capacity is lower.

If a braking system is required, it is **convenient and easy to install a brake drum or disc** on the flex coupling.

In some cases, where the driven machine cannot be rotated by hand, **maintenance procedures of oil checking and refilling**, as well as alignment, **become more difficult**.

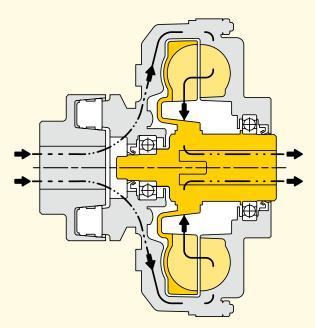
The delayed fill chamber, when present, is fitted on the driven side. The rotating speed of the said chamber gradually increases during start-up, thus **leading to a longer starting time**, assuming the bleed orifices diameters are not changed. **If oil quantity is excessively reduced**, the transmissible torque may be lower than the starting torque of the driven machine. In such a case, part of the oil remains inside the delayed chamber. This lack of oil in the fluid coupling may cause stalling.

The "**switching pin**" **device might not work correctly** on machines where, owing to irregular operating conditions, the driven side may suddenly stop or jam during the starting phase.

Flex coupling is protected by the placement of the fluid coupling before it, and therefore this configuration is fit for applications with frequent start-ups or inversions of the rotating sense.

4.2 REVERSE MOUNTING

Driver **outer** impeller



Higher inertia directly connected to the motor.

The outer impeller, being directly connected to the motor, reaches synchronous speed instantly. **Ventilation** is therefore **maximum** from the beginning.

The **assembly of a brake disc or drum** on KR fluid couplings is **more difficult, expensive** and leads to a longer axial length of the whole machine group.

The outer impeller and cover are connected to the motor, it is therefore **possible to manually rotate the coupling** to check alignment and oil level, and for refilling.

The delayed fill chamber is fitted on the driver side, and reaches the synchronous speed in a few seconds.

Oil is therefore centrifuged into the main circuit gradually and completely.

Starting time is adjustable by replacing the calibrated bleed orifices. **The starting phase**, however **is performed in a shorter time** than in the configuration with an inner driver impeller.

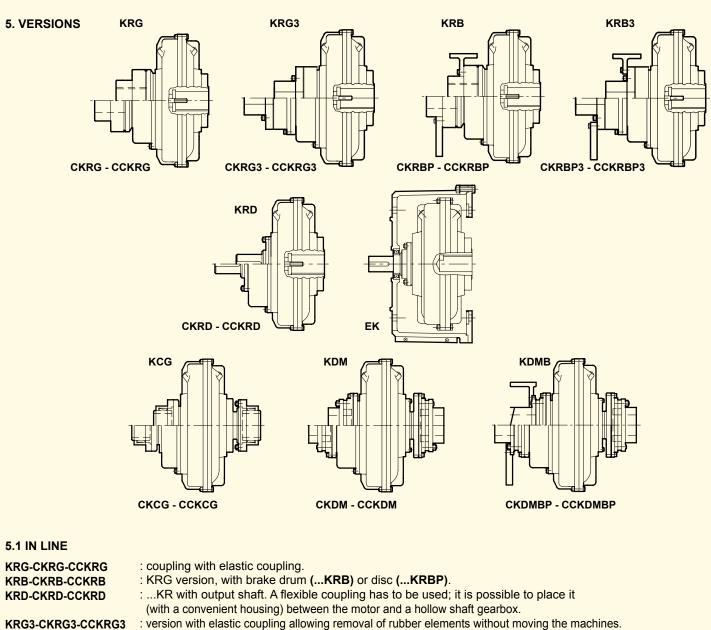
The **switching pin operation is always assured**, where fitted, as the outer impeller always rotates because it is mounted on the driver shaft.

In case of frequent start-ups or inversions of the rotating direction, the **flex coupling is much more stressed**.

If not expressely required by the customer or needed for the application being performed, the fluid coupling is supplied according to our "standard" mounting. Do specify in your request for quotation whether you need a "reverse" mounting.

NOTE: Starting from size **13K** and **11CK** included, a baffle ring is always fitted on the driver impeller, and therefore it is not recommended to mount a fluid coupling "reverse" if "standard" mounting, or viceversa. In these cases contact TRANSFLUID for more detailed information.

PRODUCTION PROGRAM



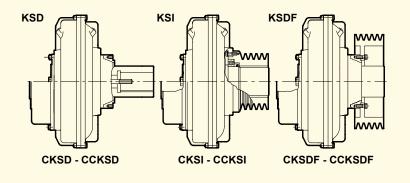
- KRM-CKRM-CCKRM : coupling with clamp type, super elastic coupling.
 - : fluid coupling fitted with a bell housing, to be placed between a flanged electric motor and a hollow shaft gearbox.
- **KCG-CKCG-CCKCG KDM-CKDM-CCKDM** : fluid coupling with gear couplings, also available with brake drum (...**KCGB**) or disc (...**KCGBP**). : fluid coupling with disc couplings, also available with brake drum (...**KDMB**) or disc (...**KDMBP**).

N.B.: The ...KCG - ...KDM versions allow a radial disassembly without moving the motor or the driven machine.

5.1 PULLEY

EΚ

KSD-CKSD-CCKSD	: basic coupling foreseen for a flanged pulley, with simple (CK) or double (CCK) delayed fill chamber.
KSI-CKSI-CCKSI	: fluid coupling with an incorporated pulley, which is fitted from inside.
KSDF-CKSDF-CCKS	: KSD coupling with flanged pulley, externally mounted and therefore to be easily disassembled.



6 MOUNTING

6.1 IN LINE VERSIONS MOUNTING EXAMPLES

- Horizontal axis between the Fig. A motor and the driven machine (KRG-CKRG-CCKRG and similar).
- It allows a radial disassembly Fig. B without moving the motor and the driven machine (KCG-KDM and similar).
- Fig. C Between a flanged electric motor and a hollow shaft gearbox by means of a bell housing (...KRD and EK).
- Fig. D Vertical axis mounting between the electric motor and a gearbox or driven machine. In case of order, please

specify mounting type 1 or 2.

Fig. E Between the motor and a upported pulley for high powers and heavy radial loads.

N.B. Version EK (fig. C) also for vertical mounting (fig. D 1-2)

6.2 PULLEY VERSIONS MOUNTING EXAMPLES

- Fig. F Horizontal axis
- Fig. G Vertical axis. When ordering, please specify mounting type 1 or 2.

7 SPECIAL VERSION

7.1 ATEX

It is possible to get the Transfluid fluid couplings with finished bores certified as equipment for intended use in hazardous zones according to directive 2014/34/UE (Atex).

The selection of suitable Atex fluid coupling must consider an additional safety factor of 1.2 times the absorbed power (for instance, motor 132 kW @ 1500 rpm-absorbed power 120 kW x 1.2 = 144 kW power to be considered in the selection).

According to different categories, there is the suitable selected fluid coupling as per below table.

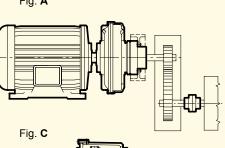
Fluid coupling model	Category 3 Ex II 3G Ex hIIC T4 Gc Ex II 3D Ex hIIIC T125°C Dc	Category 2 Ex II 2G Ex hIIC T4 Gb Ex II 2D Ex hIIIC T125°C Db	Category 1 Ex I M2 Ex h I Mb
KRG	•	•	•
KCG	•	•	
KDM	•	•	•
KXG	•	•	
KXD	•	•	•
EK	•		
KBM	•	•	
KSD	•	• (water)	
Fluid fill	Oil or Treated water	Fire resistant oil Treated water	Treated water only

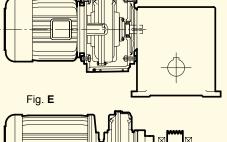
In case of inquiry for Atex fluid coupling, you have to apply Transfluid providing the application form TF 6413 duly filled up. About KXG and KXD couplings, please refer to catalogue 160 GB.

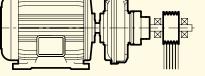
7.2 WATER FILL FLUID COUPLING

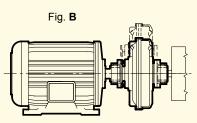
Transfluid has developed a version of water fill fluid coupling in order to meet the demands of environment friendly products as well as couplings suitable for working in hazardous zone and underground mines.

Fig. A

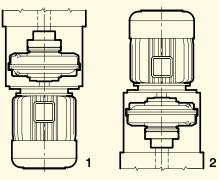


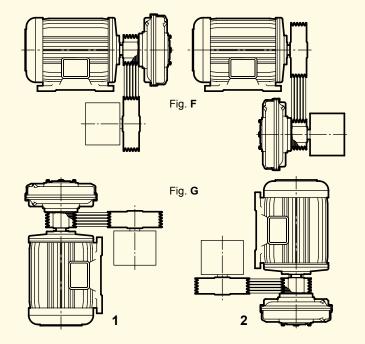












The water to be used is a mixture of water and glycole. The water fill couplings are available upon request on all design from size 13 upwards; they have the same overall dimensions of standard couplings series. A suffix "W" identifies the coupling suitable for treated water operation (e.g. 27 CKRGW)

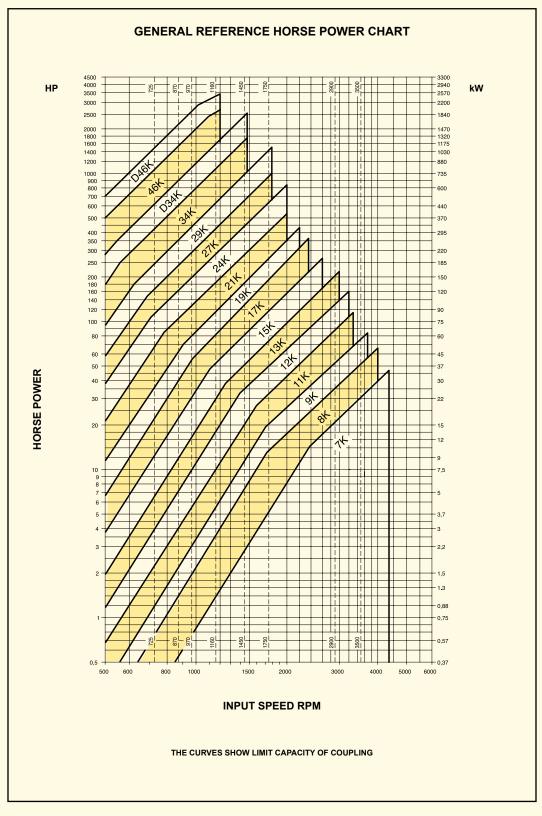
7.3 LOW TEMPERATURE (below -20°C)

KDM - KCG - Special bearings - Special seal fluid.



8 SELECTION 8.1 SELECTION CHART

The chart below may be used to select a unit size from the horsepower and input speed. If the selection point falls on a size limit line dividing one size from the other, it is advisable to select the larger size with a proportionally reduced oil fill.

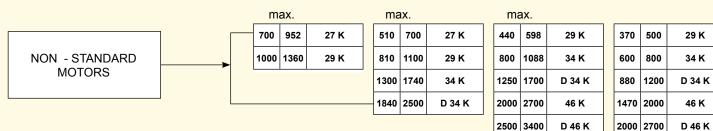


Tab. A

8.2 SELECTION TABLE

Fluid coupling for standard electric motors.

МО	TOR		300	0 rpm		180	0 rpm		150	0 rpm		120) rpm		100) rpm
TYPE	SHAFT DIA.	kW	НР	COUPLING	kW	НР	COUPLING	kW	HP	COUPLING	kW	HP	COUPLING	kW	НР	COUPLING
80	19	0.75 1.1	1 1.5		0.55 0.75	0.75 1		0.55 0.75	0.75 1	-	0.37 0.55	0.5 0.75	- 17	0.37 0.55	0.5 0.75	7 K
90S	24	1.5	2		1.1	1.5		1.1	1.5	7 K	0.75	1	7 K	0.75	1	
90L	24	2.2	3		1.5	2	7 K	1.5	2		1.1	1.5		1.1	1.5	8 K
100L	28	3	4	7K	2.2	3 4		2.2	3 4		1.5	2	8 K	1.5	2	9 K
112M	28	4	5.5		4	5.5		4	5.5	8K	2.2	3		2.2	3	
132	38	5.5 7.5	7.5 10		5.5	7.5	8 K	5.5	7.5		3	4	9 K	3	4	11 K
132M	38	7.5	10		7.5	10		7.5	10	9 K	4 5.5	5.5 7.5	11 K	4	5.5 7.5	
160M	42	11 15	15 20		11	15	9 K	11	15		7.5	10		7.5	10	12 K
160L	42	18.5	25	9K	15	20	11 K	15	20	11 K	11	15	12 K	11	15	13 K
180M	48	22	30		18.5	25	12 K (11 K)	18.5	25		-	-	-	-	-	-
180L	48	-	-	-	22	30	12 K	22	30	12 K	15	20	13 K	15	20	
200L	55	30 37	40 50	11 K	30	40	13 K (12 K)	30	40		18.5 22	25 30		18.5 22	25 30	15 K
225S	60	-	-	-	37	50		37	50	13 K	-	-	45.14	-	-	-
225M	55(300) 60	45	60	11 K	45	60	13 K	45	60		30	40	15 K	30	40	17 K
250M	60 (3000) 65	55	75	13 K	55	75	15 K	55	75	15 K	37	50		37	50	
280S	65 (3000) 75	75	100		75	100	17 K (15 K)	75	100		45	60	17 K	45	60	19 K
280M	65 (3000) 75	90	125	13 K	90	125		90	125	17 K	55	75		55	75	~
315S	65 (3000) 80	110	150		110	150	17 K	110	150	19 K	75	100	19 K	75	100	21 K
315M	65 (3000) 80	132 160	180 220	-	132 160 200	180 220 270	19 K	132 160 200	180 220 270	21 K	90 110 132	125 150 180	21 K	90 110 132	125 150 180	24 K
355S	80 (3000) 100	200	270	-	260	340	21 K	250	340	0.4 14	160	220	24 K	160	220	27 K
355M	80 (3000) 100	250	340	-	315	430	24 K	315	430	24 K	200 250	270 340	27 K	200 250	270 340	29 K



NB: THE FLUID COUPLING SIZE IS TIED TO THE MOTOR SHAFT DIMENSIONS

8.3 PERFORMANCE CALCULATIONS

For frequent starts or high inertia acceleration, it is necessary to first carry out the following calculations. For this purpose it is necessary to know:

Ρm	- input power	kW
nm	- input speed	rpm
ΡI	- power absorbed by the load at rated speed	kW
n	- speed of driven machine	rpm
J	- inertia of driven machine	kgm ²
Т	- ambient temperature	°Č

The preliminary selection will be made from the selection graph Tab. **A** depending upon input power and speed. Then check:

A) acceleration time

B) max allowable temperature

C) max working cycles per hour

A) Acceleration time ta :

$$t_a = \frac{n_u \cdot J_r}{9.55 \cdot M_a}$$
 (sec) where:

 n_{U} = coupling output speed (rpm)

 J_r = inertia of driven machine feddered to coupling shaft (kgm²) M_a = acceleration torque (Nm)

 $n_{u} = n_{m} \cdot \left(\frac{100 - S}{100}\right)$

where S is the percent slip derived from the characteristic curves of the coupling with respect to the absorbed torque M_I .

If S is not known accurately, the following assumptions may be made for initial calculations:

4 up to size 13"

3 from size 15" up to size 19"

2 for all larger sizes.

$$J_{\Gamma} = J \cdot \left(\frac{n_{L}}{n_{U}}\right)^{2}$$

Note:

 $J = \frac{PD^2}{4} \text{ or } \frac{GD^2}{4}$

 $M_{a} = 1.65 \text{ M}_{m} - \text{M}_{L}$ where: $M_{m} = \frac{9550 \cdot \text{P}_{m}}{\text{N}_{m}}$ (Nominal Torque) $M_{L} = \frac{9550 \cdot \text{P}_{L}}{\text{N}_{U}}$ (Absorbed Torque)

B) Max allowable temperature.

For simplicity of calculation, ignore the heat dissipated during acceleration.

Coupling temperature rise during start-up is given by:

$$T_a = \frac{Q}{C}$$
 (°C)

where: Q = heat generated during acceleration (kcal)

C = total thermal capacity (metal and oil) of coupling selected from Tab. **C** (kcal/°C).

$$Q = \frac{n_{U}}{10^{4}} \cdot \left(\frac{J_{\Gamma} \cdot n_{U}}{76.5} + \frac{M_{L} \cdot t_{a}}{8}\right) \text{ (kcal)}$$

The final coupling temperature reached at the end of the acceleration cycle will be:

$$T_f = T + T_a + T_L (°C)$$

where: T_f = final temperature (°C)

T = ambient temperature (°C) T_a = temperature rise during acceleration (°C) T_L = temperature during steady running (°C)

$$T_{L} = 2.4 \cdot \frac{P_{L} \cdot S}{K} \quad (^{\circ}C)$$

where: K = factor from Tab. **D** Tf = must not exceed 150° C

C) Max working cycles per hour H

In addition to the heat generated in the coupling by slip during steady running, heat is also generated (as calculated above) during the acceleration period. To allow time for this heat to be dissipated, one must not exceed the max allowable number of acceleration cycles per hour.

H max =
$$\frac{3600}{t_a + t_L}$$

where t_I = minimum working time

$$r_{\rm L} = 10^3 \frac{\rm Q}{\left(\frac{t_a}{2} + T_{\rm L}\right) \cdot \kappa}$$
 (sec)

t

SELECTION

8.4 CALCULATION EXAMPLE

Assuming: Pm = 20 kW nm = 1450 giri/min PL = 12 kW n_L = 700 giri/min $J = 350 \text{ kgm}^2$ T = 25 °C

Trasmission via belts.

From selection graph. on Tab. A, selected size is 12K.

A) Acceleration time

From curve Tf 5078-X (supplied on request) slip S = 4%

$$n_{\rm u} = 1450 \cdot \left(\frac{100 - 4}{100}\right) = 1392 \text{ rpm}$$

$$J_{\rm r} = 350 \cdot \left(\frac{700}{1392}\right)^2 = 88.5 \text{ kgm}^2$$

$$M_{\rm m} = \frac{9550 \cdot 20}{1450} = 131 \text{ Nm}$$

$$M_{\rm L} = \frac{9550 \cdot 12}{1392} = 82 \text{ Nm}$$

 $M_a = 1,65 \cdot 131 - 82 = 134 \text{ Nm}$

$$t_a = \frac{1392 \cdot 88.5}{9.55 \cdot 134} = 96 \text{ sec}$$

B) Max allowable temperature

$$Q = \frac{1392}{10^4} \cdot \left(\frac{88.5 \cdot 1392}{76.5} + \frac{82 \cdot 96}{8}\right) = 361 \text{ kcal}$$

$$C = 4.2 \text{ kcal/°C (Tab.C)}$$

$$T_a = \frac{361}{4.2} = 86 ^{\circ}C$$

$$K = 8.9 \text{ (Tab. D)}$$

$$T_L = 2.4 \cdot \frac{12 \cdot 4}{8.9} = 13 ^{\circ}C$$

$$T_4 = 25 + 86 + 13 = 124 ^{\circ}C$$

C) Max working cycles per hour

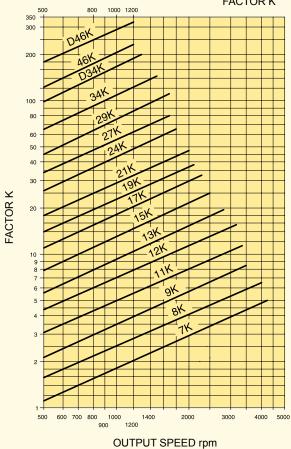
$$t_{L} = 10^{3} \cdot \frac{361}{\left(\frac{86}{2} + 13\right) \cdot 8.9} = 724 \text{ sec}$$

H = $\frac{3600}{96 + 724} = 4 \text{ starts per hour}$

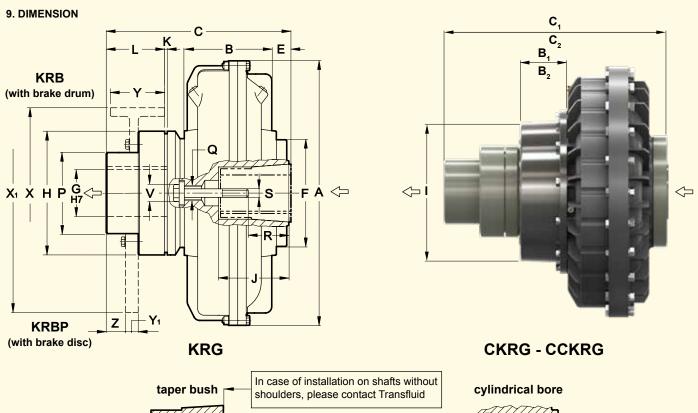
Tab. **C** THERMAL CAPACITY

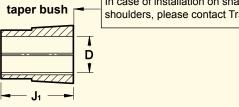
size	K kcal/°C	CK kcal/°C	CCK kcal/°C
7	1.2		
8	1.5	-	
9	2.5		
11	3.2	3.7	-
12	4.2	5	
13	6	6.8	
15	9	10	10.3
17	12.8	14.6	15.8
19	15.4	17.3	19.4
21	21.8	25.4	27.5
24	29	32	33.8
27	43	50	53.9
29	56	63	66.6
34	92	99	101
D34	138	-	-
46	-	-	175
D46	332	-	-

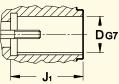


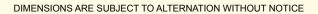


SERIES 7 ÷ 19 - KRG - KRB - KRBP - CK... - CCK...









1	Size	DH>	Dime	ensic	ns																														
Ċ		[)	J	J ₁		Α	B KR	B 1 CKR	B22 CCKR	C KRG	C ₁ CKRG	C22 CCKRG			G max	Н	I	K	L	Ρ	Q	R	S	V	Z	Flex coupling	brake drum X - Y	brake disc X ₁ - Y ₁	(wit	eight k hout c CKRG	oil)		Oil nax (I CKRG	
	7	19	24		40 5	0	228	77			189			22									27 35	M6 M8						8.3			0.92		
	'	2	8	69	60		220				109				114	42	110			60	70	M12	40	M10	21		BT 10	160 - 60		0.5			0.92		
	8	2	4	00	50	;	256	91	-		194	_		18				-			10		36	M8	- '		51 10	100 00		8.7	_		1.5	-	
	Ŭ	2			60			0.															41	M10											
	9	28	38		60 8	- 1	295	96			246			31										M10 M12		-			on request	16			1.95		
	-	42•••			80 11					-			-		128				2				79	M16								-			-
1	1	28	38	111	60 8		325	107	68.5			301		27		55	132	195			85	M20			27		BT 20	160 - 60 200 - 75		18	20.5		2.75	3.35	
	_	42••• 28			80 11 60 8		_				255									80			83	M16				200 - 75							
1	2	42	38 49		80 8	- 1	372	122				322		24	145								42 56 83	M10 M12 M16						21.5	24.5		4.1	4.8	
\vdash	_	42	40 •• 48		110				75									224					84	M16	\vdash			000 75	400.00						
1	3	55		143	110 58	5	398	137			285	345		28	177	70					100		74 104			5	BT 30	200 - 75 250 - 95	400 - 30	34	37		5.2	5.8	
\vdash		48	55		110		_										170							M16 M20	34			250 - 95	400 - 30						
1	5		65	145	140		460	151	87	137	343	411	461	35	206	80		259		110	120		100	M20		35	BT40	315 - 118	450 - 30	50.3	54.3	62	7.65	8.6	9.3
		48	55		110																		80	M16 M20											
1	7	60	65•••	145	140	-	520	170						37					3			M27	103							77	83	92	11.7	13.6	14.9
	Ì	75•	80•	-	140 17	70				470		440	500			~	050	0.07		110	105		103 133	M20		45	DT 50	315 - 118	445 - 30						
		48	55	145	110				96	1/6	302	442	522		225	90	250	337		110	135		80	M16 M20] ³⁴	15	BT 50	400 - 150	450 - 30						
1	9	60	65 …	140	140		565	190						17									103	M20						83	90	99	14.2	16.5	18.5
		75•	80•	-	140 17	70																	103 133	10120											

 D BORES RELATIVE TO TAPER BUSHES WITH A KEYWAY ACCORDING TO ISO 773 - DIN 6885/1 PARTICULAR CASES:

• CYLINDRICAL BORE WITHOUT TAPER BUSH WITH A KEYWAY ISO 773 - DIN 6885/1

• CYLINDRICAL BORE WITHOUT TAPER BUSH, WITH A REDUCED KEYWAY (DIN 6885/2)

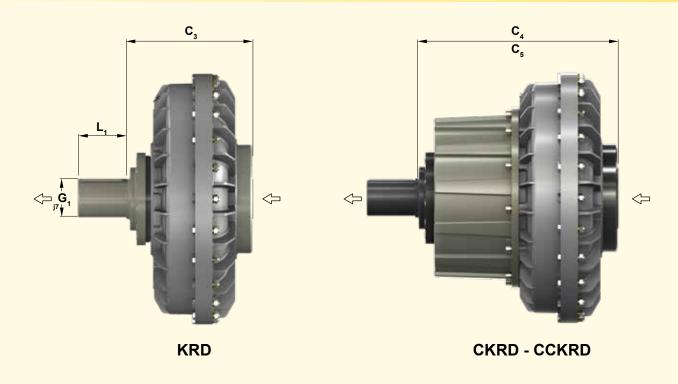
••• TAPER BUSH WITHOUT KEYWAY

- FOR ... KRB - KRBP SERIES SPECIFY X AND Y OR X1 AND Y1 DIAMETER

EXAMPLE: 9KRB - D38 - BRAKE DRUM = 160x60

SERIES 7 ÷ 19 - KRD - CKRD - CCKRD





NB: The arrows \triangleleft indicate input and output in the standard versions.

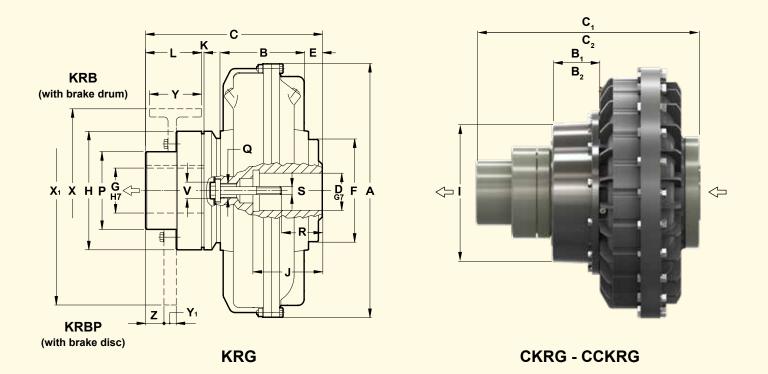
	$\Sigma \Rightarrow$	Dime	nsion	s				
AN Size	l .	C₄ ckrd	C₅ cckrd	G ₁	L ₁	(wi	eight thout CKRD	-
7	138			28	40	5.7		
8	138	-		20	40	6.1	-	
9	176			38		11.6		
11	185	231	-	42	50	13	15.5	-
12	100	252		42		16.7	19.7	
13	212	272		48	60	26.3	29.3	
15	230	298	348	60	80	40.4	44.4	52.1
17	236	242	423	75	100	58.1	64.1	73.1
19	230	343	423	15	100	65.1	71.1	80.1

– WHEN ORDERING, SPECIFY: SIZE - MODEL - ${\bf D}$ DIAMETER – UPON REQUEST: BORE ${\bf G}$ MACHINED; ${\bf G}1$ SPECIAL SHAFT

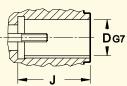
- G1 SHAFT WITH A KEYWAY ACCORDING TO ISO 773 DIN 6885/1



SERIES 21 ÷ 34 - KRG - KRB - KRBP - CK... - CCK...



cylindrical bore



NB: The arrows \triangleleft indicate input and output in the standard versions.

e	Σ	X> I	Dime	nsio	ns																												
AT Size		D)	J	A	B KR	B₁ CKR	B ₂ CCKR	C KR	C ₁ CKR	C₂ CCKR	Е	F	G	н	I	к	L	Р	Q	R	S	v		Flex coupling	Brake drum X - Y	Brake disc X ₁ - Y ₁	(wit	eight thout CKRG	oil)		Oil nax I ^{CKRG}	
21	•	•80	90	170	620	205			433	533	623	45									130	M20 M24					560 - 30	129	139	147	19	23	31
21		••1	00	210	020	205	110	199	468	568	658	80	250	110	290	400	3	140	170	M36	165	M24	10	15	BT60	400 - 150	630 - 30	129	139	147	19	23	31
24	•	•80	90	170		229		199	433	533	623	21	200	110	230	400	5	140	170	10130	130	M20 M24	40	45	10100	500 - 190		147	157	165	40	31.2	20
24		••1	00	210	/ 14	229			468	568	658	56									165	M24					795 - 30	147	157	105	40	31.2	39
27	,	120 r	max	210 max	780	278			484	602	702	15	315	130	354		4	150	200		167 (for	M24 max bore)	-	20	BTSO	500 - 190	710 - 30	228	246	265	42	50	61
29)	135 ı	max	240 max	860	295	131	231	513	631	731	18	350	150	554	537	4	150	200	M45	167 (for	M24 max bore)	-	20	БТОО	500 - 190	795 - 30	281	299	309	55	63	73
34	ŀ	150 r	nax i	265 max	1000	368			638	749	849	19	400	140	395		5	170	220		200 (for	M36 max bore)	-	18	ВТ90	630 - 236	1000 - 30	472	482	496	82.5	92.5	101

D BORES WITH A KEYWAY ACCORDING TO ISO 773 - DIN 6885/1 _

STANDARD DIMENSIONS WITH A KEYWAY ISO 773 - DIN 6885/1 •

•• STANDARD DIMENSIONS WITH REDUCED KEYWAY (DIN 6885/2)

_

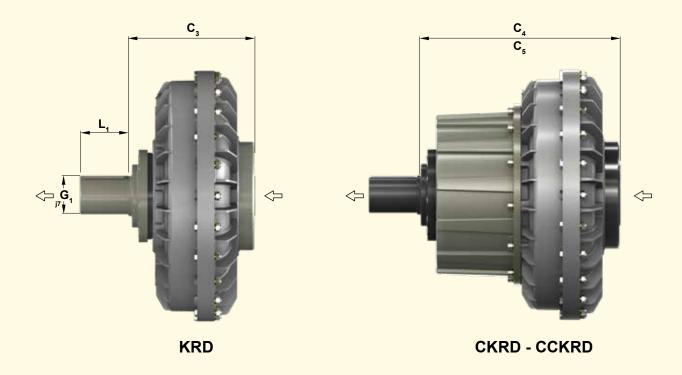
WHEN ORDERING, SPECIFY: SIZE - MODEL- D DIAMETER. FOR ...KRB OR ...KRBP, SPECIFY X AND Y OR X1 AND Y1 DIMENSIONS BRAKE DRUM OR DISC

UPON REQUEST, G FINISHED BORE

EXAMPLE: 19KRBP - D80 - BRAKE DISC 450 x 30

SERIES 21 ÷ 34 - KRD - CKRD - CCKRD





NB: The arrows \triangleleft indicate input and output in the standard versions.

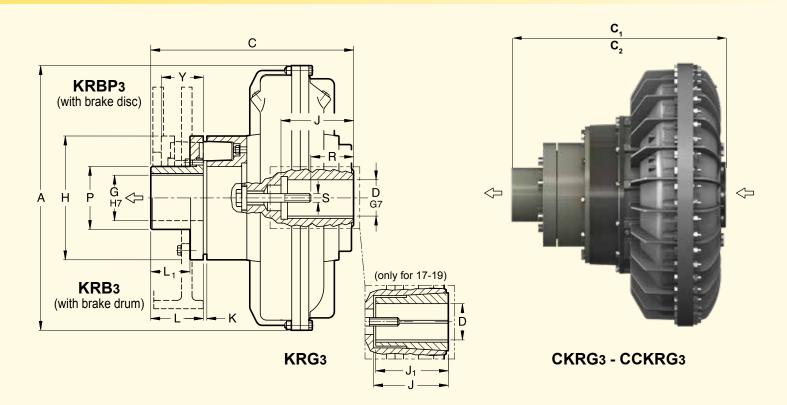
	∑≒> D	imensio	ons					
⊠ Size			C₅ CCKRD	G ₁	L,		eight thout	
ų	RND	CRED	CONND			KRD	CKRD	CCKRD
21	292	392	482			99.5	109.5	117.5
21	327*	427*	517*		120	99.5	109.5	117.5
24	292	292 392		90	120	447 5	407.5	405.5
24	327*					117.5	127.5	135.5
27	333	451	551	100	140	178	186	215
29	362	480	580	100	140	231	249	259
34	437	568	668	140	150	358	373	383

*

Total lenght with D100 UPON REQUEST \mathbf{G}_1 SPECIAL SHAFT DIAMETER _

DIMENSIONS ARE SUBJECT TO ALTERATION WITHOUT NOTICE

SERIES 17 ÷ 46 - KRG3 - KRBP - CK... - CCK...



The three pieces flexible coupling B3T allows the removal of the elastic elements (rubber blocks), without removal of the electric motor; only with the ...KRB3 (with brake drum) coupling the electric motor must be moved by the value of 'Y'. 'Y' = axial displacement male part of the coupling B3T necessary for the removal of the elastic elements.

	$\Sigma \not\Rightarrow \rangle$	Dimen	sions																				
de Size	I	D	J	J ₁	A	с	C ₁	C ₂	G	н	к	L	L,	Р	R	ł	ę	6	Y	Elastic coupling	(V	Veight k /ithout c CKRG3	g bil) CCKRG₃
	48	55	445	110											80	C	M16	M20					
17	60	65•••	145	140	520										10	3	м	20			84	90	99
	75• 80•	-	140 - 170		418	498	578	90	240		110	82	120	103	132	1 171	20	82	D2T 50				
			145	110		410	490	5/6	90	240	3		02	130	80	C	M16	M20	02	B3T-50			
19	48 55	140	140	565										10	3	м	20			91	97	106	
	75•	80•	-	140 - 170											103	132		20					

D BORES RELEVANT TO TAPER BUSH WITH KEYWAY ACCORDING TO ISO773 - DIN6885/1 _

STANDARD CYLINDRICAL BORES WITHOUT TAPER BUSH WITH KEYWAY ACCORDING TO ISO773 - DIN6885/1

··· TAPER BUSH WITHOUT KEYWAY

21	80•	90	170		620	457	557	647							130	M20	M24			134	144	152
21	100	••	210]	620	492	592	682	110	290	3	140	78	150	165	M	24	82	B3T-60	134	144	152
24	80•	90	170]	714	457	557	647	110	290		140	10	150	130	M20	M24	02	B31-00	152	162	170
24	100	••	210		/ 14	492	592	682							165	M	24			152	102	170
27	120 ו	max	210	-	780	566	684	784	130	254		150	112	180	167	M	24	120	D2T 00	247	265	284
29	135 ו	max	240		860	595	713	813	130	354	4	150	112	160	for ma	ax hole		120	B3T-80	300	318	328
34	150 ו	max	265		1000	704	815	915	150	395	5	170	119	205	200 for ma	M3 Ax hole	6	151	B3T-90	505	481	491
46	180 ו	max	320	-	1330	-	-	1092	180	490	7	195	138	270	190 for ma		6	122	B3T-100	-	-	1102

D CYLINDRICAL BORES WITHOUT TAPER BUSH WITH KEYWAY ACCORDING TO ISO773 - DIN6885/1

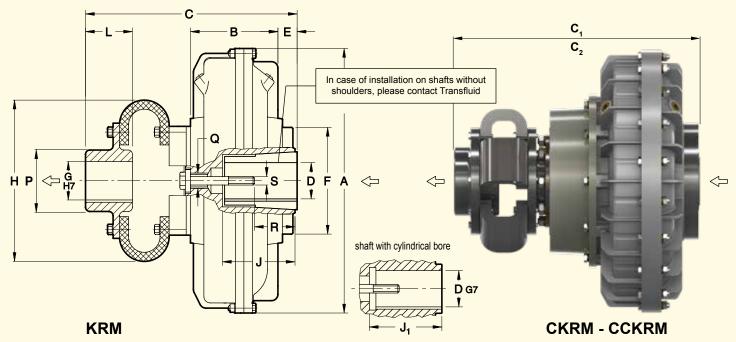
STANDARD DIMENSIONS

••

STANDARD DIMENSION WITH REDUCED KEYWAY (DIN 6885/2) ON ORDER FORM PLEASE SPECIFY: DIMENSION, MODEL, DIAMETER D - EXAMPLE: 21CCKRG3 - D80 _

SERIES 9 ÷ 34 - KRM - CKRM - CCKRM

industrial & marine



COUPLING ALLOWING HIGHER MISALIGNMENTS AND THE REPLACEMENT OF THE ELASTIC ELEMENTS WITHOUT MOVING THE MACHINES

TAPER BUSH VERSION ∑⇒> Dimensions Size Weight kg D J J₁ Α в С C, **C**₂ Е F G н L Ρ Q R S Elastic (without oil) Ŋ coupling KRM CKRM CCKRM KRM |CKRM |CCKRM M10 M12 _ 14.5 42••• M16 M10 M12 M20 53 F 16.5 42••• 48•• M16 M12 42••• 48•• M16 55 F 60••• 110 58.5 M20 55••• M16 M20 56 F 59.7 65••• M20 M16 M20 M27 65••• M20 140 170 105 135 75. 80. -58 F M16 M20 65... M20 140 170 105 135 75. 80. M20 -

– D BORES RELEVANT TO TAPER BUSH WITH KEYWAY ACCORDING TO ISO773 - DIN6885/1

CYLINDRICAL BORES WITHOUT TAPER BUSH WITH KEYWAY ACCORDING TO ISO773 - DIN6885/1

CYLINDRICAL BORES WITHOUT TAPER BUSH WITH A REDUCED KEYWAY (DIN 6885/2)
 TAPER BUSH WITHOUT KEYWAY

•	• IAP	ER BUSH WIT	HOUT	KEYWAY					CY	LINDR		BORE	VERS	SION									
	21	80• 90		170	620	205	496	596	686	45							130	M20	M24		124	134	142
	21	100••		210	020	205	531	631	721	80	250	90	378	110	144	M36	165	М	24	65 F	124	134	142
	24	80• 90		170	715	229	496	596	686	21	230	50	570		144	10130	130	M20	M24	031	142	152	160
	24	100••		210	/15	223	531	631	721	56							165	М	24		172	152	100
	27	120 max	-	210	780	278	525	643	743	15	315	100	462	122	160		167 (for ma		24)	66 F	211	229	248
	29	135 max		240	860	295	577	659	795	18	350	120	530	145	192	M45	167 (for ma	M x bore		68 F	293	311	321
	34	150 max		265	1000	368	648	779	879	19	400	140	630	165	224		200 (for ma		36)	610 F	467	462	492

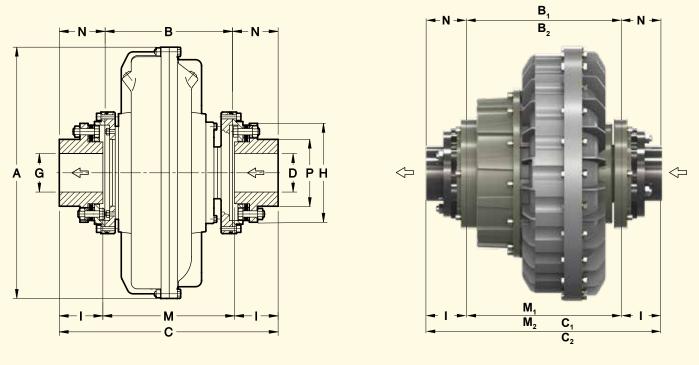
- D BORES WITH A KEYWAY ACCORDING TO ISO 773 - DIN 6885/1

STANDARD DIMENSIONS WITH A KEYWAY ISO 773 - DIN 6885/1
 STANDARD DIMENSIONS WITH REDUCED KEYWAY (DIN 6885/2)

STANDARD DIMENSIONS WITH REDUCED KEYWAY (DIN 6885/2)
 WHEN ORDERING SPECIEV: SIZE - SERIE D DIAMETER - EXAMPLE: 13 CKRI

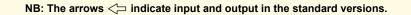
- WHEN ORDERING, SPECIFY: SIZE - SERIE D DIAMETER - EXAMPLE: 13 CKRM-D 55

SERIES 11 ÷ 34 - KDM - CKDM - CCKDM



KDM

CKDM - CCKDM



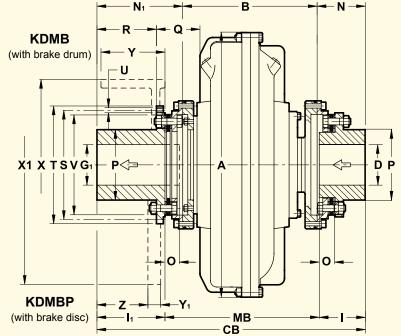
FLUID COUPLING FITTED WITH HALF DISC COUPLINGS, WITHOUT MAINTENANCE AND PRESCRIBED FOR PARTICULAR AMBIENT CONDITIONS. TO BE RADIALLY DISASSEMBLED WITHOUT MOVING THE MACHINES.

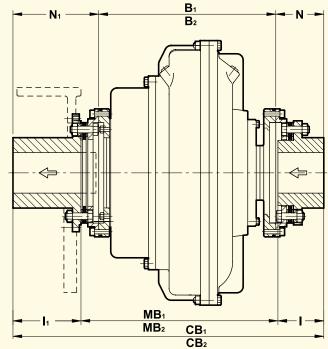
	∑\$\$ C	Dimensi	ons																	
<hd> Size</hd>	A	B KDM	В ₁ скрм	В ₂ сском	C KDM	С ₁ ском	C ₂ CCKDM	D G min	D G max	н	I	M KDM	M ₁ CKDM	M ₂ CCKDM	N	Ρ	Disc coupling size		Weight k without o CKDM	-
11	325	186	232		289	335		16	55	123	50	189	235		51.5	76	1055	22.5	25	
12	372	100	253	-	209	356	-	10	55	123	50	109	256	-	51.5	70	1055	26	29	-
13	398	216	276		339	399		21	65	147	60	219	279		61.5	88	1065	41.3	44.3	
15	460	246	314	364	391	459	509	21	75	166	70	251	319	369	72.5	104	1075	65	69	76.7
17	520	269	349	429	444	524	604	31	90	192	85	274	354	434	87.5	122	1085	89	95	104
19	565	209	349	429	444	524	004	31	90	192	00	2/4	304	434	07.5	122	1065	96	102	111
21	620	315	415	505	540	640	730	41	115	244	110	320	420	510	112.5	154	1110	159	169	177
24	714	315	415	505	540	640	730	41	115	244	110	320	420	510	112.5	154	1110	177	187	195
27	780	358	476	576	644	762	862	51	135	303	140	364	482	582	143	196	1140	289	307	326
29	860	387	505	605	673	791	891	51	135	303	140	393	511	611	143	190	1140	342	360	370
34	1000	442	573	673	768	899	999	61	165	340	160	448	579	679	163	228	1160	556	562	572

- WHEN ODERING, SPECIFY: SIZE - MODEL - FINISHED D-G BORES UPON REQUEST

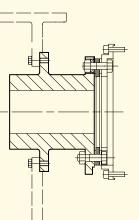
EXAMPLE: 27 CKDM







NB: The arrows \triangleleft indicate input and output in the standard versions.



∑⊰> Dimensions

<pre>CH⊂ Size</pre>	Brake drum X - Y	Brake disc X ₁ - Y ₁	(with	eight out oil, n and o CKD	brake
12	200 - 75	on request	27	30	
13	200 - 75	on request	42.5	45.8	-
15	250 - 95	450 - 30	69.3	73.3	81
17	315 - 118	500 - 30	99	105	114
19	400 - 150	560 - 30	105	112	125
21	400 - 150	630 - 30	179	189	197
24	500 - 190	710 - 30	197	207	215
27	500 - 190	800 - 30	317	335	354
29	500 - 190	000 - 30	370	388	398
34	on request	800 - 30 1000 - 30	599	587	597

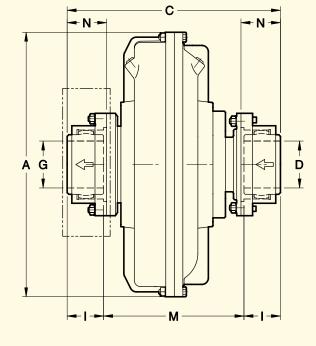
ONLY FOR 27 - 29 ARE AVAILABLE HUBS FOR BRAKE DRUM/DISC WITH CENTRAL FLANGE

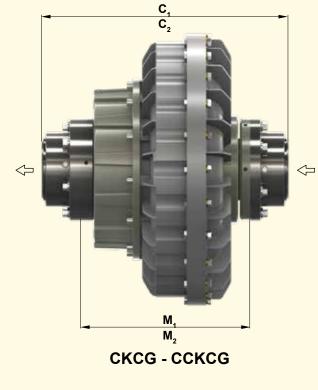
	$\Sigma \not\Rightarrow \\$	Dime	ension	IS																								
\supset Size	A	в	B1	B ₂	СВ	CB1	CB2	D	G1	I	I	1	MB	MB ₁	MB ₂	N	N ₁	ο	Р	Q	R	s	т	ι	J	v	z	Disc coupling
~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~		KDM	CKDM	CCKDM	KD	CKD	CCKDM	max	max		std	max	KD	CKD	CCKDM		St					÷0.1	f7	Nr.	Ø			size
12	372	186	253		336.5	403.5		55	60	50	8	0	206.5	273.5		51.5	99	17.5	76	67	69	128	142	8	M8	114		1055
13	398	216	276	-	440.5	500.5	-	65	65	60	140	170	240.5	300.5	-	61.5	163	21.5	88	78	129	155	170		IVIO	140	-	1065
15	460	246	314	364	495.5	563.5	613.5	75	80	70	150	170	275.5	343.5	393.5	72.5	177	24.5	104	98	134	175	192			157	109	1075
17	520	269	349	420	E 4 9 E	628.5	708.5	90	95	85		210	303.5	383.5	463.5	87.5	192	29.5	122	107	143	204	224		M10	185	118	1085
19	565	209	349	429	546.5	020.5	706.5	90	95	65	160	210	303.5	363.5	403.5	07.5	192	29.5	122	87	143	204	224			100	110	1065
21	620	315	415	505	620 5	720 5	818.5	115	120	110	100		358.5	458.5	548.5	112 5	201	38.5	154	133	137	256	276	12	M12	234	112	1110
24	714	315	415	505	020.5	/20.5	010.0	115	120	110			306.0	400.0	546.5	112.5	201	36.5	104	109	137	250	270		IVITZ	234	112	1110
27	780	358	476	576	731.5	849.5	949.5	135	145	140		240	411.5	529.5	629.5	142	230.5	47.5	100	107	155	215	220		M14	286	133	1140
29	860	387	505	605	760.5	878.5	978.5	135	145	140	180		440.5	558.5	658.5	143	230.5	47.5	196	109	155	315	338		11/14	200	133	1140
34	1000	442	573	673	845.5	976.5	1076.5	165	175	160			505.5	636.5	736.5	163	240.5	57.5	228	124	152	356	382		M16	325	130	1160

- WHEN ORDERING, SPECIFY: SIZE - MODEL

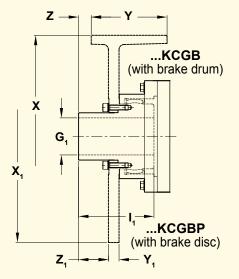
D AND G1 FINISHED BORES UPON REQUEST, AND SPECIAL I1 DIMENSION
 FOR BRAKE DRUM OR DISC, SPECIFY DIMENSIONS X AND Y OR X1 AND Y1 EXAMPLE : 17KDMB - BRAKE DRUM 400 x 150

## SERIES 7 ÷ 46 - KCG - KCGB - CKCGBP - CCKCG...- CCKCG





KCG



Brake drum or disc upon request

FLUID COUPLING FITTED WITH HALF GEAR COUPLINGS, TO BE RADIALLY
DISASSEMBLED WITHOUT MOVING THE MACHINES

	∑ <del>\</del> }	Dim	ensior	าร																
AS Size	A	С	C1	C2	D G	G1	I	۱ ₁	M	<b>M</b> 1	M2	N	Brake drum X - Y	z	Brake disc	<b>Z</b> 1	Gear Couplig Size	(wit	eight thout	oil)
			CKUG	CCKCG	max	max			-	CKCG	CCKCG		A-1		X ₁ - Y ₁				CKGC	CCKCG
7	228	229			50	-	43	80	143			44.5	•	•		•	1" E.I.	11.3		
8	256	234	-						148	-							(5)(6)	11.7	-	
9	295	290.6							190.6									22.9		
11	325	299.6	345.6	-	65	45	50	114	199.6	245.6	-	50.8	250-95	45	400-30	32	1" ½ E.I.	24.9	27.4	-
12	372	299.6	366.6		65	45	50	114	199.0	266.6		50.6	250-95	45	400-30	32	(5) (6)	28.5	31.4	
13	398	325.1	385.6						225.1	285.1	]							37.6	40.6	
15	460	410	478	528					258	326	376		050.05				2" 1/2	76.6	80.6	88.3
17	520	434	514	594	95	65	76	146	282	362	442	79.5	250-95 315-118	57.5 21.5	400-30 445-30	44.5	E.I.	91.1	97.1	106.1
19	565	434	514	594					202	302	442		515-116	21.5	445-50		(5) (6)	98.1	104.1	113.1
21	620	503	603	693	111	90	90	165	323	423	513	93.5	315-118	26	560-30	38	3" E.I.	142.3	152.3	160.3
24	714	503	603	693	m	90	90	105	323	423	513	93.5	400-150	15	710-30	38	(5) (6)	160.3	170.3	178.3
27	780	627	754	845	124	110	105	170	417	535	635	109.5	500-190	6	795-30	30	3" ½ E.I.	253.2	272.2	291.2
29	860	656	774	874	154	110	105	170	446	564	664	109.5	500-190	0	795-30	30	(5) (6)	307.2	325.2	335.2
34	1000	750	881	981	160	120	120	190	510	641	741	123.5	•	•	800-30	42	4" E.I. (5) (6)	492.4	507.4	517.4
46	1330	-	-	1313.4	244	175	190	280	-	-	933.4	192.5	•	•	•	•	6" E.I. (5) (6)	-	-	1333

UPON REQUEST

(5) E.I. = EXPOSED INCH SCREWS

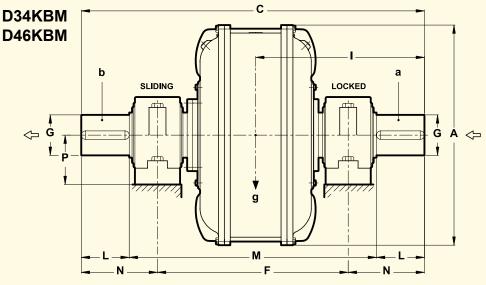
(6) GEAR COUPLING WITH SPECIAL CALIBRATED BOLTS

- WHEN ORDERING, SPECIFY: SIZE - MODEL

EXAMPLE: 21CKCG



FLUID COUPLING WITH DOUBLE CIRCUIT, FITTED WITH MAIN SUPPORTS AND INPUT AND OUTPUT SHAFTS



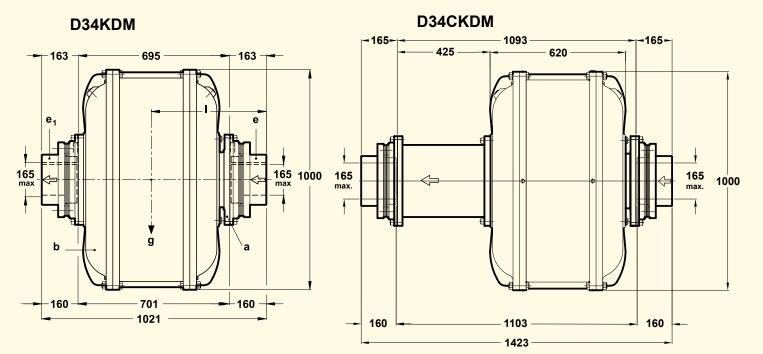
SERIES	A	С	F	D-G m6	L	М	N	Ρ	 WEIGHT Kg (Without oil)	OIL max. I	CENTER O g Kg	F GRAVITY I mm	MOMENT C J (WR2 a	
D34KBM	1000	1400	855	140	140	1120	257.5	170	810	162	952	710	26.19	64.25
D46KBM	1330	1900	1275	160	200	1550	312.5	170	2200	390	2514	955	91.25	183.7

KEYWAYS ACCORDING TO ISO 773 - DIN 6885/1

FLUID COUPLINGS FITTED WITH DOUBLE CIRCUIT, TO BE RADIALLY DISASSEMBLED WITHOUT MOVING THE MACHINES.

WITH HALF DISC COUPLINGS, WITHOUT MAINTENANCE

WITH HALF DISC COUPLINGS, WITHOUT MAINTENANCE

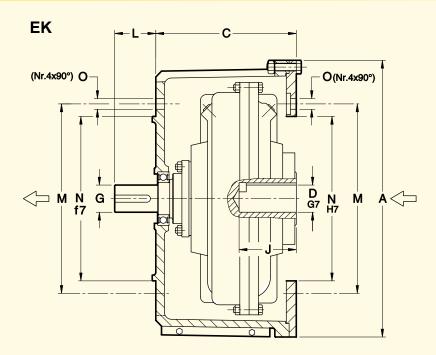


NB: The arrows <> indicate input and output in the standard versions.

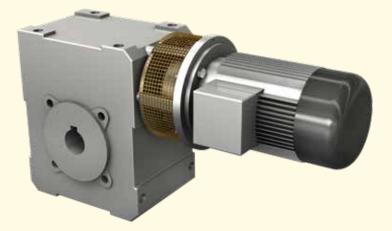
e	∑≍> Dime	nsions						
M	WEIGHT Kg (without oil)	OIL	CENTER O g	1			OF INE ) Kgm	
77	(miniout on)		kg	mm	а	b	С	d
D34KDM	880	162	1022	512	26.08	65.53	0.955	0.955
D34CKDM	1014	194.5	194.5	532	26.08	67.99	0.955	0.955

Also available D46KCG. For information please apply Transfluid

- g = TOTAL WEIGHT INCLUDING OIL (MAX FILL)
- a = INTERNAL ELEMENT
- b = EXTERNAL ELEMENT
- d = HALF FLEXIBLE COUPLING (INTERNAL ELEMENT)
- d₁ = HALF FLEXIBLE COUPLING (EXTERNAL ELEMENT)



Example for application



NB: The arrows  $\triangleleft \neg$  indicate input and output in the standard versions.

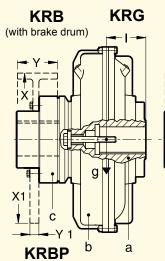
	e	∑≍> C	Dimens	sions										
[	AL Size	D	J	G	L	A	с	м	N	0	Weight kg (without oil)	OIL max I	Electric TYPE	motors kW 1500 rpm
	7	• 24	52	24	38	269	132	165	130	11	11.4	0.92	90S - 90L **90LL	1.1 -1.5 1.6
	8	• 28	62	28 h7	44	299	142	215	180	13	12.2	1.5	100 L 112 M	2.2 -3 4
	9	• 38	82	38	57	399	187	265	230	13	26.9	1.95	132S - 132M ** 132L	5.5 - 7.5 9.2
1	1	• 42	112	42	63	399	187	300	250	17	28.3	2.75	160M -160 L	11 - 15
1	2	•• 48	112	48 j7	65	485	214	300	250	17	66	4.1	180 M 180 L	18.5 22
1	3	• 55	112	55	80	400	214	350	300	17	76	5.2	200 L	30

CYLINDRICAL BORE WITH A KEYWAY ISO 773 - DIN 6885/1

CYLINDRICAL BORE WITH A REDUCED KEYWAY (DIN 6885/2)
 NOT STANDARD

WHEN ORDERING SPECIFY: SIZE - MODEL - DIAMETER D and G EXAMPLE: 8 EK-D28 - G 28

## CENTER OF GRAVITY MOMENT OF INERTIA



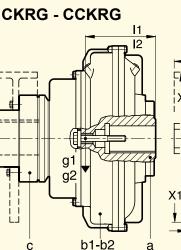
(with brake disc)

**KDM** 

Y

e1

X1



**CKDM-CCKDM** 

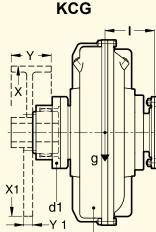
e1

e

g1

g2

b1-b2



b

е

11

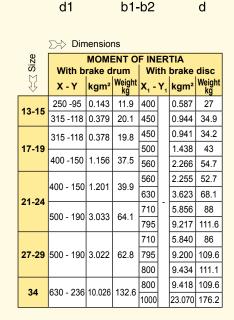
12

d



| a1

g2



**MOMENT OF INERTIA J kgm²** 

С

0.004

0.011

0.032

0.082

0.192

0.370

1.350

3.185

6.68

b₂

-

0.887

1.372

1.879

3.181

5.420

10.37

13.754

29.983

106.6

b1

.

0.217

0.359

0.601

1.281

1.788

2.997

5.236

9.410

13.126

а

0.006

0.012

0.020

0.039

0.072

0.122

0.236

0.465

0.770

1.244

2.546

3.278

4.750

11.950

52.2

b

0.019

0.034

0.068

0.109

0.189

0.307

0.591

1.025

1.533

2.407

4.646

7.353

11.070

.KRG

...ĸcg

d۱

0.004

0.016

0.102

0.102

0.375

0.436

1.649

7.14

b = EXTERNAL ELEMENT + COVER

b2 = b + DOUBLE DELAY CHAMBER

d

0.004

0.017

0.091

0.091

0.145

0.500

0.798

4.35

...KDM

e1

-

0.014

0.036

0.064

0.125

0.373

0.887

2.773

е

-

0.014

0.032

0.063

0.121

0.210

0.934

1.565

#### ∑⇒> Dimensions

g

h

Size							(	ENT	ER O	F GR	AVIT	(						
	KF	RG	СК	RG	CCK	RG	K	G	СК	CG	CCF	CG	K	М	СК	DM	CCK	(DM
¥	g Kg	l mm	g₁ Kg	l ₁ mm	g₂ Kg	l ₂ mm	g Kg	l mm	g₁ Kg	l₁ mm	g₂ Kg	l₂ mm	g Kg	l mm	g₁ Kg	l₁ mm	g₂ Kg	l₂ mm
7	9.1	92					12.1	70										
8	10	93	-	-			13	73	-	-			-	-	-	-		
9	17.7	134					24.6	86					22.2	81				
11	20.4	136	23.4	151	-	-	27.3	93	30.2	107	1 -	-	24.9	85	27.9	98	-	-
12	25.1	142	28.7	154			32.1	98	35.6	113			29.6	92	33.2	104		
13	38.5	157	42	176			42.2	104	45.7	115			45.8	101	49.3	109		
15	57	174	61.8	195	70.2	216	77.3	124	82.1	135	90.4	147	71.7	121.5	76.6	130	85.7	145
17	87.2	205	94.8	225	106.5	238	85.3	138	103.1	152	126.6	185	99.2	135	106.9	145	118.3	163
19	96.4	201	104.4	221	116	227	104.6		112.6	152	136	182	106.4		116.4	145	127.4	161
21	145.6	233	159	265	169.3	288	151.2	157	164.5	174	200.2	211	175.6	156	189	168	201	182
24	172	227	184	255	195.3	280	177.2		190.2	170	225.2	201	202	150	214.3	166	226	178
27	265	262	290	298	313	312	276.2	185	304.2	210	361.2	248	326	164	351	174	378	195
29			305	368	321	344.2	198	359.2	218	415.2	251	383	176	411	188	432	200	
34	521	333	549	364	580	376	548.9	235	571.9	253	582.9	282	628	209	636	214	650	222
46			-		1294	485			-		1524	368				-		

g g1 g2 = TOTAL WEIGHT, INCLUDING OIL (MAX FILL)

* For **KSD** (without pulley) = a + b

* For CKSD (without pulley) = a + b1

* For CCKSD (without pulley) = a + b2

a = INTERNAL ELEMENT b1 = b + DELAY CHAMBER

27.299 29.356

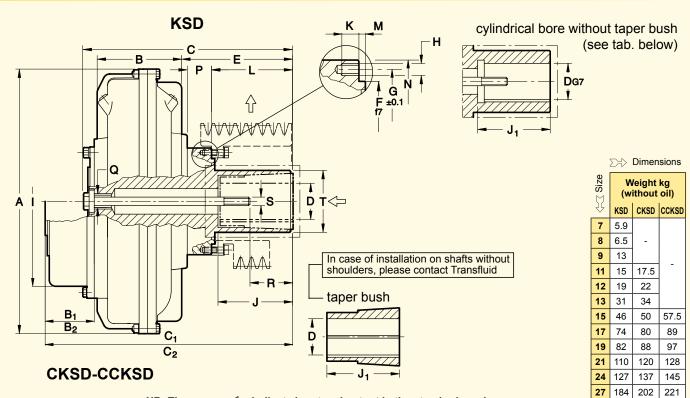
c FLEXIBLE COUPLING

d e = HALF FLEXIBLE COUPLING (INTERNAL ELEMENT)

d1 e1 = HALF FLEXIBLE COUPLING (EXTERNAL ELEMENT) EXAMPLE: J..CCKCG = a+d (INT. ELEM.) b2+d1 (EXT. ELEM.)



## SERIES 7 ÷ 27 - KSD - CKSD - CCKSD



NB: The arrows <> indicate input and output in the standard versions.

	$\Sigma \Rightarrow$	Dimen	nsions										TAF	PER B	USH	VERS	ION										
Size		C	J	J	1	Α	в	B ₁	B ₂	с	С ₁	C ₂	Е	F	G	H	ł	Т	к	L	м	N	Ρ	Q	R	S	т
Ŋ							KSD	CKSD	сскзр	max	CKSD	сскѕр				Nr.	ø										max
7	19	24		40	50	228	77			159			55							35					29 3	8 M6 N	8
'	2	8	69	6	0	220				174			70	75	90	4	M6		8	50	3	114	14	M12	43	M10	50
8	2	4	09	5	0	256	91	_		194	_		81	15	50	-		_		65	Ŭ			10112	33	M8	
Ŭ	2	8		6	0	200	01			104			01												43	M10	
9	28	38		60	80	295	96			250			116												39 6		12
	•••	42	111	8	0				-			-	-	96	114					85	5	128	20		78	M16	69
11	28	38		60	80	325	107	73.5		259	289.5		113			8		195						M20	38 5		12
	•••	•42		8	0		-						-				M8		13						78	M16	$\rightarrow$
12	38	42	113	80	110	372	122			274	327		125	112	130					98	7	145	22		54 8		80
		•48		11				80									-	224							83	M16	
13	42	48	144	11		398	137			367	407		190	135	155					158	6	177	29		76	M16	88
																										06 M20	
15	48	55	145	11		460	151	92	142	390	438	488	195	150	178			264	17	159		206	28		80 7		100
	60	•••65		14	-																				100	M20	_
4-	48	55	145	11			470						045			12							60	M27	69 99	_	
17	60	•••65		14		520	170						245				M10				7		60			39	
	•75	48 55 14	-	140	170			101	181	455	516	596		180	200			337	17	180		225			69	M20	132
10			145	11	-	565	190						225										45		99		
19	60 •75	•••65 •80		140		505	190						225										40			39	
	•/5	•80	-	140	170																				39 1	55	

D BORES RELATIVE TO TAPER BUSHES WITH A KEYWAY ACCORDING TO ISO 773 - DIN 6885/1 PARTICUALR CASES:

CYLINDRICAL BORE WITHOUT TAPER BUSH ISO 773 - DIN 6885/1

TAPER BUSH WITHOUT A KEYWAY •••

#### CYLINDRICAL BORE VERSION

														-												
21	•80			170	620	205			505	580	670	260							190			57		135	M20	
21	•100			210	620	205	115	205	545	620	710	300	200	228	8	M14	400	23	230	7	250	57	M46	165	M24	145
24	•80		-	170	714	229		205	505	580	670	236	200	220	°	1114	400	23	190	/	250	M46	10140	135	M20	145
24	•100			210	/ 14	229			545	620	710	276							230			10140		165	M24	
27	120 ma	x		210	780	278	138									CONS	SULT O	UR EN	IGINEE	RS						

STANDARD CYLINDRICAL BORES WITH KEYWAYS ACCORDING TO ISO 773 - DIN 6885/1

WHEN ORDERING SPECIFY: SIZE - MODEL - D DIAMETER

EXAMPLE: 12KSD - D 42

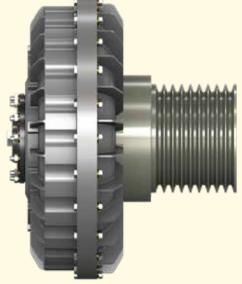
CONSULT OUR ENGINEERS

## STANDARD PULLEYS

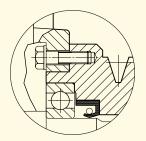


U

## **KSI - CKSI - CCKSI**

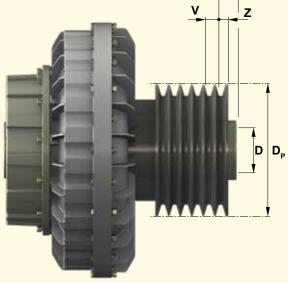


...KSI

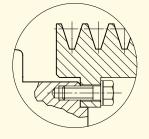


_	

**KSDF - CKSDF - CCKSDF** 



...KSDF



∑⇒> Dimensions

And Size	D	U	Flanged pulley	
Y			Dp	N° type
7	19 - 24	6	125	
1	28	21	125	2 - SPA/A
8	19 - 24	36	125	
0	28	9	112	3 - SPA/A
9	28 - 38	34	160	4 - SPB/B
11	42	58	200	3 - SPB/B
	38 - 42	50	180	4 - SPB/B
12	30 - 42 48	51	200	3 - SPC/C
		26		4 - SPC/C
	42 - 48 55 - 60	12.5	180	6 - SPB/B
13		50 49	250	6 - SPB/B 5 - SPC/C
		12.5	200	6 - SPB/B
15	48 - 55 60 - 65	17	250	5 - SPC/C
	00 00	69	280	5 - SPB/B
		72.5	280	6 - SPB/B
17	67 - 75 80	85.5	310	6 - SPC/C
19		72.5	315	6 - SPB/B
		59	345	6 - SPC/C
21 24	Upon request			
27				

	∑⊰> Dimensions					
	Size	D	D U		Integral pulley	
	Y			Dp	N° type	
Γ	7	19 - 24	11.5	80		
				90	2 - SPA/A	
				100		
		-	26.5	80		
				90		
				100		
	8	19 - 24	26.5	90	3 - SPA/A	
	0	13 24		100		
	9	38	10	112	5 - SPA/A	
	11	42	15	125	4 - SPB/B	
	12	38 - 42 48	12	140	5 - SPB/B	

GROOVE	v	z
SPZ/Z	12	8
SPA/A	15	10
SPB/B	19	12.5
SPC/C	25.5	17
D	37	24
3 V	10.3	8.7
5 V	17.5	12.7
8 V	28.6	19

- WHEN ORDERING, SPECIFY: SIZE - MODEL - D DIAMETER - Dp - NUMBER AND TYPE OF GROOVES EXAMPLE: 13 CKSDF - D55 - PULLEY Dp. 250 - 5 SPC/C



### 10. FILLING

Transfluid hydraulic couplings are supplied without oil. Standard filling: X for K series, 2 for CK series, and 3 for CCK series. The quantities are indicated on page 13 and 15 of this catalog. Follow the procedure indicated on Installation and Maintenance manuals 150 GB and 155 GB delivered with each coupling. Suggested oil: **ISO32 HM** for normal operating temperatures. For temperatures down zero, **ISO FD 10 (SAE 5W)** and for

temperatures lower than -20°C contact TRANSFLUID.

## **11. SAFETY DEVICES**

### **FUSIBLE PLUG**

In case of overloads, or when slip reaches very high values, oil temperature increases excessively, damaging oil seals and conseguently allowing leakage.

To avoid damage when used in severe applications, it is advisable to fit a fusible plug. Fluid couplings are supplied with a fusible plug at  $140^{\circ}$ C ( $109^{\circ}$ C,  $120^{\circ}$ C or  $198^{\circ}$ C upon request).

#### **SWITCHING PIN**

Oil venting from fusible plug may be avoided with the installation of a switching pin. When the temperature reaches the melting point of the fusible ring element, a pin releases that intercepts a relay cam that can be used for an alarm or stopping the main motor. As for the fusible plug, 2 different fusible rings are available (below).

#### **11.1 SWITCHING PIN DEVICE**

This device includes a percussion fusible plug installed on the taper plug. The percussion fusible plug is made of a threaded plug and a pin hold by a fusible ring coming out due to the centrifugal force when the foreseen melting temperature is reached. Such increase of temperature can be due to overload, machinery blockage or insufficient oil filling. The pin, moving by approx. 16 mm, intercepts the cam of the switch to operate an alarm or motor trip signal.

After a possible intervention and removal of the producing reason, this device can be easily restored with the replacement of the percussion plug or even the fusible ring following the specific instructions included in the instruction manual.

With external wheel as driver, as indicated in Fig. 5, the percussion plug operates in any condition, while in case of driven external wheel it can operate correctly only in case of increase of the slip due to overload or excessive absorption.

It is possible to install this system on all fluid couplings starting from size 13K even in case it has not been included as initial supply, asking for a kit including percussion fusible plug, gasket, modified taper plug, counterweight for balancing, glue, lever switch assembly installation instructions.

In order to increase the safety of the fluid coupling a standard fusible plug is always installed, set at a temperature higher than that of the percussion fusible plug.

For a correct operation, please refer to the instructions relevant to the standard or reverse installation described at page 6.

- Lever switch standard supply 230 Vac
- Upon request: Atex version
- Switching pin available: see below tab

M12x1.5 7.5 ch 19	MELTIN	G TEMPER	ATURE ^{+10°C} 0
	109°C	SPEC.	GA1004D
	120°C	SPEC.	GA1004A
	140°C	SPEC.	GA1004B
1 🕈			

#### ELECTRONIC OVERLOAD CONTROLLER

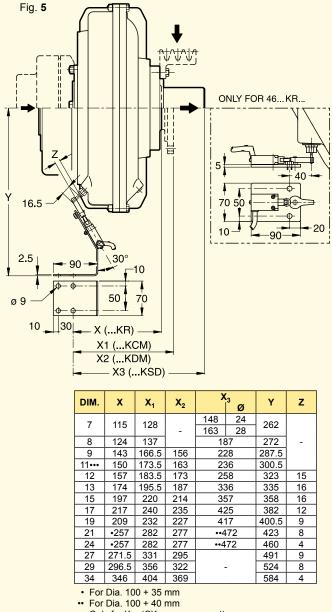
This device consists of a proximity sensors measuring the speed variation between the input and output of the fluid coupling and giving an alarm signal or stopping the motor in case the set threshold is overcome.

With such a device, as well as with the infrared temperature controller, no further maintenance or repair intervention is necessary after the overload occupance, because the machinery can operate normally, once the cause of the inconvenience has been removed (see page 28).

## INFRARED TEMPERATURE CONTROLLER

To measure the operating temperature, a device fitted with an infrared sensor is available. After conveniently positioning it by the fluid coupling, it allows a very precise non-contact temperature measurement.

Temperature values are reported on a display that also allows the setting of 2 alarm thresholds, that can be used by the customer (see page 29).



••• Only for K... (CK... upon request)

REFERENCE DIMENSIONS

## 11.2 OVERLOAD CONTROLLER (Fig. 6)

When load torque increases, slip also increases and output speed consequently decreases.

The said speed variation can be measured by means of a sensor sending a pulse train to the speed controller. If the rotating speed goes lower than the set threshold (see diagram) on the controller, a signal is given through the intervention of the inner relay.

The device has a "TC" timer with a blind time before starting (1 -120 s) avoiding the alarm intervention during the starting phase, and another "T" timer (1 – 30 s) preventing from undesired relay intervention during sudden changes of torque.

The device also provides a speed proportional analogic output signal (0 – 10 V), that can be forwarded to a display or a signal transducer (4 - 20 mA).

Standard supply is 230 V ac, other supplies are available upon request: 115 V ac, 24 V ac or 24 V dc, to be specified with the order.

Atex version is available too.

CONTROLLER PANEL (Fig. 7)

(TC) Blind time for starting

Set screw regulation up to 120 s

#### (DS) Speed range regulation

Programmable DIP-SWITCH (5 positions), selecting relay status, proximity type, reset system, acceleration or deceleration. Programming speed Dip-Switch with 6 positions allows to choose the most suitable speed range, according to the application being performed.

Speed level (set point) (SP)

Set screw regulation with digits from 0 to 10. The value 10 corresponds to full range set with DS1 Dip-Switch.

#### (R) Reset

Local manual reset is possible through R button, or remote reset by connecting a N.O. contact at pins 2-10.

#### (SS)Threshold overtaking

(RED LED) It lights up every time that the set threshold (set point) is overtaken.

#### Alarm led (A)

(RED LED) It lights up when alarm is ON and the inner relay is closed.

#### (E) Enable

(YELLOW LED) It lights up when the device is enabled.

(т)

## **Delay time**

Set screw regulation up to 30 s.

#### (ON) Supply

(GREEN LED) It shows that the device is electrically supplied.

FOR FURTHER DETAILS ASK FOR TE 5800-A



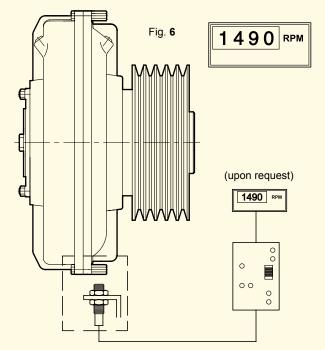
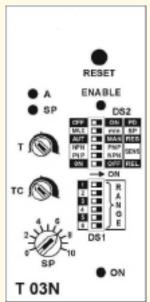
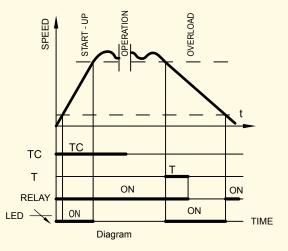


Fig. 7





alarm.

## **11.3 INFRARED TEMPERATURE CONTROLLER**

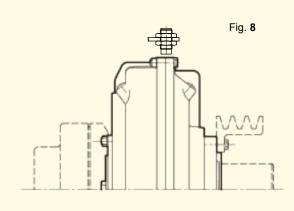
This is a non contact system used to check fluid coupling temperature. It is reliable and easily mounted. It has 2 adjustable thresholds with one logical alarm and one relay

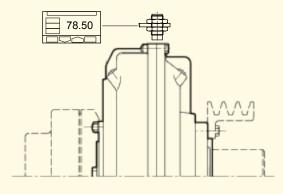
The proximity sensor must be positioned near the fluid coupling outer impeller or cover, according to one of the layouts shown in Fig.  $\mathbf{8}$ .

The distance between the sensor and the fluid coupling must be about 15-20 mm (cooling fins do not disturb the correct operation of the sensor).

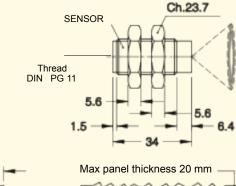
To avoid that the bright surface of the fluid coupling reflects light, and thus compromises a correct temperature reading, it is necessary to paint the surface, directly facing the sensor with a flat black colour (a stripe of 6-7 cm is sufficient).

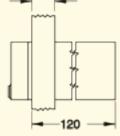
The sensor cable has a standard length of 90 cm. If required, a longer one may be used only if plaited and shielded as per type "K" thermocouples.

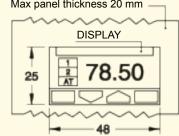




SENSOR				
Temperature range	0 ÷ 200 °C			
Ambient temperature	- 18 ÷ 70 °C			
Accuracy	0.0001 °C			
Dimensions	32.5 x 20 mm			
Standard wire lenght •	0.9 m			
Body	ABS			
Protection	IP 65			
CONTROLLER				
Power supply	85264 Vac / 4863 Hz			
Relay output OP1	No (2A - 250 V)			
Logical output OP2	Not insulated			
(5Vdc, ±10%, 30 mA max)				
AL1 alarm (display)	Logic (OP2)			
AL2 alarm (display)	Relay (OP1) (NO, 2A / 250Vac)			
Pins protection	IP 20			
Body protection	IP 30			
Display protection	IP 65			
Dimensions	1/32 DIN – 48x24x120 mm			
Weight	100 gr			







• CAN BE MADE LONGER WITH TWISTED AND SHIELDED WIRES FOR TYPE K THERMOCOUPLES (NOT SUPPLIED)

## OTHER TRANSFLUID PRODUCTS FOR ELECTRIC MOTOR APPLICATION



## FLUID COUPLING KSL SERIES

Start up and variable speed drive up to 4000 kW



FLEXIBLE COUPLING BM-B3M SERIES Up to 33100 Nm



## PNEUMATIC CLUTCH TP SERIES

Up to 16800 Nm



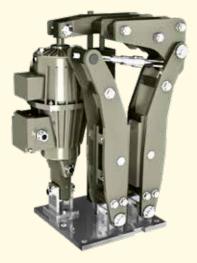
## FLUID COUPLING KPT SERIES

Start up and variable speed drive up to 1700 kW



DISC & DRUM BRAKE NBG/TFDS SERIES

Up to 19000 Nm



ELECTRIC MACHINES PERMANENT MAGNETS SYNCHRONOUS AC

Up to 100 kW





TRANSFLUE industrial & mari